AMENDMENTS TO THE CLAIMS

- 1. (Currently Amended) A refrigerator having comprising:
- a first compressor for compressing a coolant,
- a first radiator for radiating heat from the first coolant,
- a <u>first</u> flow control valve for regulating the flow volume of the <u>first</u> coolant,
- and an a first evaporator for evaporating the first coolant, characterized in that the refrigerator includes:
 - a-coolant cooling means for cooling the first coolant; and
- a-heat-exchange-amount control means for controlling the amount quantity of heat exchanged in the coolant cooling means, wherein the <u>first</u> coolant is circulated through the <u>first</u> compressor, the <u>first</u> radiator, the coolant cooling means, the <u>first</u> flow control valve, and the <u>first</u> evaporator, in that sequence.
- 2. (Currently Amended) AThe refrigerator as recited claimed in claim 1, utilizing a nonflammable coolant whose having a global warming potential is lower than that of chlorofluorocarbon, wherein the coolant cooling means includes:
- a second compressor for compressing a second coolant whose having an energy consumption efficiency is higher than that of the first coolant;
 - a condenser for radiating heat from the second coolant;
 - a second flow control valve for regulating the flow volume of the second coolant; and
- a second evaporator for evaporating, by means of with heat from the first coolant, the second coolant; wherein the second coolant is circulated through the second compressor, the condenser, the second flow control valve, and the second evaporator, in that sequence.
- 3. (Currently Amended) AThe refrigerator as recited in claim 1, wherein the compressor having comprises an intermediary-pressure inlet for drawing in the first coolant during compressing, the refrigerator further comprising:
- a gas-liquid separator for separating into gas and liquid the <u>first</u> coolantas outputtedoutput from the <u>first</u> flow control valve;
- a bypass pipe for introducing into the intermediary-pressure inlet <u>at least</u> part or all of <u>the gas of</u> the <u>first</u> coolant gas-separated by the gas-liquid separator; and
- a third flow control valve for regulating the flow volume of the <u>first</u> coolant-as outputted<u>output</u> from the gas-liquid separator and <u>inputted</u>input into the <u>first</u> evaporator.

- 4. (Currently Amended) <u>AThe</u> refrigerator as <u>recitedclaimed</u> in claim 1, further comprising:
- a thirdsecond compressor for compressing the first coolant as-compressed by the first compressor;
- a gas-liquid separator for separating into gas and liquid the <u>first</u> coolant as outputtedoutput from the <u>first</u> flow control valve;
- a bypass pipe for introducing into the thirdsecond compressor at least part or all of the gas of the first coolant gas-separated by the gas-liquid separator; and
- a third second flow control valve for regulating the flow volume of the first coolant as outputted output from the gas-liquid separator and inputted input into the first evaporator; wherein the first coolant as outputted output from the third second compressor is inputted input into the first radiator.
- 5. (Currently Amended) AThe refrigerator as recited claimed in claim 1, further comprising:
- a thirdsecond radiator for radiating heat from the first coolant as outputted output from the first compressor; and
- a <u>third</u>second compressor for compressing the <u>first</u> coolant in a state in which heat of the <u>first</u> coolant has been radiated away by the <u>third</u>second radiator; wherein the <u>first</u> coolant <u>is flowedflows</u> through the <u>third</u>second radiator, the <u>third</u>second compressor, and the <u>first</u> radiator, in that sequence.
- 6. (Currently Amended) AThe refrigerator as recited claimed in claim 2, further comprising:
- a third compressor for compressing the $\underline{\text{first}}$ coolant $\underline{\text{as-}}$ compressed by the $\underline{\text{first}}$ compressor; and
- a third-heat exchanger for exchanging heat between the <u>first</u> coolant and the second coolant₃, wherein
- the <u>first</u> coolant <u>as outputted output</u> from the <u>first</u> compressor <u>is flowed flows</u> through the <u>third</u>-heat <u>exchange</u> the third compressor, and the <u>first</u> radiator, in that sequence, and
- the second coolant as outputted output from the second evaporator is flowed flows through the third-heat exchanger, and the second compressor, in that sequence.

- 7. (Currently Amended) AThe refrigerator as recited claimed in claim 2, further comprising:
- a third compressor for compressing the $\underline{\text{first}}$ coolant $\underline{\text{as-}}$ compressed by the $\underline{\text{first}}$ compressor;
- a third-heat exchanger for exchanging heat between the first coolant and the second coolant; and
- a forththird flow control valve for regulating the flow volume of the second coolant flowing in the third heat exchanger; wherein
- the <u>first</u> coolant as outputtedoutput from the <u>first</u> compressor is flowed flows through the third heat exchanger, the third compressor, and the <u>first</u> radiator, in that sequence, and
- part of the second coolant as outputted output from the condenser is flowed flows through the forth third flow control valve, the third heat exchanger, and the second compressor, in that sequence.
- 8. (Currently Amended) AThe refrigerator as recited claimed in claim 1, wherein the heat-exchange-amount control means includes:
- a-drying-ratio estimation means for estimating, by a measured value using a predetermined-sensor, a drying ratio that is a ratio-between a-drying rate of the first coolant at the exit of exiting the first flow control valve and a-drying rate when the first coolant at the exit of exiting the first radiator is decompressed to its evaporation temperature;
- and drying-ratio control-range determination means for determining a control range of the drying ratio, so that a <u>coefficient of performance (COP)</u> value is obtained, in which the difference between the <u>COP</u> value and the maximum <u>COP</u> value obtained when the drying ratio is varied under predetermined operational conditions is within a predetermined range; and
- a-control means for controlling the amount quantity of heat exchanged in the coolant cooling means, so that the drying ratio estimated by the drying-ratio estimation means is within the control range.
- 9. (Currently Amended) AThe refrigerator as recited claimed in claim 2, wherein the heat-exchange-amount control means includes:
- a-drying-ratio estimation means for estimating, by a measured value using a predetermined sensor, a drying ratio that is a ratio-between a drying rate of the first coolant at the exit of exiting the first flow control valve and a drying rate when the first coolant at the exit of exiting the first radiator is decompressed to its evaporation temperature;

a-drying-ratio control-range determination means for determining a control range of the drying ratio, so that a coefficient of performance (COP) value is obtained, in which the difference between the <u>COP</u> value and the maximum <u>COP</u> value obtained when the drying ratio is varied under predetermined operational conditions is within a predetermined range; and

a-control means for controlling the flow volume of the second coolant flowing in the coolant cooling means, so that the drying ratio estimated by the drying-ratio estimation means is within the control range.

10. (Currently Amended) A The refrigerator as recited claimed in either-claim 8-or elaim 9, wherein the predetermined-sensor includes:

at least one of a-first pressure-measuring means for measuring pressure of the <u>first</u> coolant between the exit of exiting the <u>first</u> flow control valve and the entrance entering of the evaporator, and a-first temperature-measuring means for measuring temperature of the <u>first</u> coolant at the exit of exiting the <u>first</u> flow control valve;

æ-second pressure-measuring means for measuring pressure of the <u>first</u> coolant between the <u>first</u> compressor and the <u>first</u> flow control valve;

at the entrance of entering the first flow control valve; and

a-third temperature-measuring means for measuring temperature of the <u>first</u> coolant at the exit of exiting the <u>first</u> radiator.

- 11. (Currently Amended) A<u>The</u> refrigerator as recited claimed in either claim 8-or elaim-9, wherein the predetermined sensor includes:
- e-first temperature-measuring means for measuring temperature of the <u>first</u> coolant et the exit of exiting the <u>first</u> flow control valve;
- a-second temperature-measuring means for measuring temperature of the <u>first</u> coolant at the entrance of entering the <u>first</u> flow control valve;
- a-third temperature-measuring means for measuring temperature of the <u>first</u> coolant at the exit-of exiting the <u>first</u> radiator;
- a forth fourth temperature-measuring means for measuring temperature of the first coolant at the entrance of entering the first radiator; and
- e-fifth temperature-measuring means for measuring temperature of the <u>first</u> coolant et the entrance of entering the <u>first</u> compressor.

12.(Currently Amended) AThe refrigerator as recited claimed in claim 1, further comprising:

entrance temperature as evolunt-temperature at the entrance of the first coolant entering the first flow control valve, wherein the heat-exchange-amount control means includes:

#flow-control-valve-entrance-temperature control-range determination means for determining #control range of the first flow-control-valve entrance temperature, so that a coefficient of performance (COP) value is obtained, in which the difference between the COP value and the maximum COP value obtained when the flow-control-valve entrance temperature is varied under predetermined operational conditions is within a predetermined range; and

a-control means for controlling the amount quantity of heat exchanged in the coolant cooling means, so that the coolant-temperature of the first coolant measured by the second first temperature-measuring means is within the control range.

13. (Currently Amended) <u>AThe</u> refrigerator as <u>recited</u> in claim 2, further comprising:

a-second first temperature-measuring means for measuring first flow-control-valve entrance temperature as evolunt-temperature at the entrance of the first coolant entering the first flow control valve; wherein the heat-exchange-amount control means includes:

#flow-control-valve-entrance-temperature control-range determination means for determining #control range of the first flow-control-valve entrance temperature, so that a coefficient of performance (COP) value is obtained, in which the difference between the COP value and the maximum COP value obtained when the first flow-control-valve entrance temperature is varied under predetermined operational conditions is within a predetermined range; and

a-control means for controlling the flow volume of the second coolant flowing in the coolant cooling means, so that the ecolant-temperature of the first coolant measured by the second-temperature-measuring means is within the control range.

14. (Currently Amended) AThe refrigerator as recited claimed in claim 1, further comprising:

a-third first temperature-measuring means for measuring coolant temperature at the exit of the first coolant exiting the first radiator; wherein the heat-exchange-amount control means includes:

a-flow-control-valve-entrance-temperature estimation means for estimating, by

from the temperature measured by the third first temperature-measuring means and the amount quantity of heat exchanged in the coolant cooling means, flow-control-valve entrance temperature as the coolant temperature at the entrance-of the first coolant entering the first flow control valve;

#flow-control-valve-entrance-temperature control-range determination means for determining a control range of the flow-control-valve entrance temperature, so that a coefficient of performance (COP) value is obtained, in which the difference between the COP value and the maximum COP value obtained when the flow-control-valve entrance temperature is varied under predetermined operational conditions is within a predetermined range; and

a=control means for controlling the amount quantity of heat exchanged in the coolant cooling means, so that the flow-control-valve entrance temperature estimated by the flow-control-valve-entrance-temperature estimation means is within the control range.

15. (Currently Amended) <u>AThe</u> refrigerator as <u>recited</u> claimed in claim 2, further comprising:

a third-temperature-measuring means for measuring coolant temperature at the exit-of coolant exiting the first radiator; wherein the heat-exchange-amount control means includes:

a-flow-control-valve-entrance-temperature estimation means for estimating, by from the temperature measured by the third-temperature-measuring means and the amount quantity of heat exchanged in the coolant cooling means, temperature at the entrance of coolant entering the first flow control valve as evolunt temperature at the entrance of the flow-control-valve entrance temperature;

#flow-control-valve-entrance-temperature control-range determination means for determining a control range of the flow-control-valve entrance temperature, so that a coefficient of performance (COP) value is obtained, in which the difference between the value and the maximum COP value obtained when the flow-control-valve entrance temperature is COP varied under predetermined operational conditions is within a predetermined range; and

e-control means for controlling the-flow volume of the second coolant flowing in the coolant cooling means, so that the flow-control-valve entrance temperature estimated by the flow-control-valve-entrance-temperature estimation means is within the control range.

16. (Currently Amended) AThe refrigerator as recited claimed in either-claim 8-or elaim-9, further comprising at least eithera first one of pressure-measuring means for measuring pressure of the first coolant between the exit of exiting the first flow control valve

and the entrance of entering the first evaporator, or a first and temperature-measuring means for measuring temperature of the first coolant at the exit of exiting the first flow control valve; wherein the drying-ratio control-range determination means determines a control range of the drying ratio, using either the evaluate pressure of the first coolant measured by the first pressure-measuring means or the evaluate temperature of the first coolant measured by the first temperature-measuring means.

- 17. (Currently Amended) AThe refrigerator as recited claimed in either-claim 8-or elaim-9, further comprising: a second-pressure-measuring means for measuring pressure of the first coolant between the exit of exiting the first radiator and the entrance of entering the first flow control valves, wherein the drying-ratio control-range determination means determines a control range of the drying ratio, using the evolunt-pressure of the first coolant measured by the second-pressure-measuring means.
- 18. (Currently Amended) AThe refrigerator as recited claimed in any-one of claims 14 to 17 claim 12, further comprising at least one of the first-pressure-measuring means for measuring pressure of the first coolant between the exit of exiting the first flow control valve and the entrance of entering the first evaporator, and the first second temperature-measuring means for measuring temperature of the first coolant at the exit of exiting the first flow control valve, wherein the flow-control-valve-entrance-temperature control-range determination means determines a control range of the temperature at the entrance of the first coolant entering the first flow control valve, using either the evolant pressure of the first coolant measured by the first-pressure-measuring means or the evolant-temperature measured by the first-second temperature-measuring means.
- 19. (Currently Amended) AThe refrigerator as recited claimed in any-one of claims 14 to 17 claim 12, further comprising: a second-pressure-measuring means for measuring pressure of the first coolant between the exitexiting of the first radiator and the entrance of entering the first flow control valve; wherein the flow-control-valve-entrance-temperature control-range determination means determines a control range of the temperature atof the entrance of first coolant entering the first flow control valve, using the coolant pressure of the first coolant measured by the second-pressure-measuring means.
 - 20. (Currently Amended) An air conditioner having comprising:
 - a first compressor for compressing a first coolant,
 - a first four-way valve for switching the-direction in which the first coolant as

eutputtedoutput from the first compressor flows,

an outdoor heat exchanger for exchanging heat between the <u>first</u> coolant and outdoor air₅,

a <u>first</u> flow control valve for regulating the flow volume of the <u>first</u> coolant, and, an indoor heat exchanger for exchanging heat between the <u>first</u> coolant and indoor air, characterized in that the air conditioner includes:

a-coolant cooling/heating means for cooling as well as and heating the first coolant; and

a-heat-exchange-amount control means for controlling the amount quantity of heat exchanged in the coolant cooling/heating means; wherein

when the air conditioner is being operated for cooling, the <u>first</u> coolant is circulated through the <u>first</u> compressor, the outdoor heat exchanger, the coolant cooling/heating means, the <u>first</u> flow control valve, and the indoor heat exchanger, in that sequence, and

when the air conditioner is being operated for warming, the <u>first</u> coolant is circulated through the <u>first</u> compressor, the indoor heat exchanger, the <u>first</u> flow control valve, the coolant cooling/heating means, and the outdoor heat exchanger, in that sequence.

- 21. (Currently Amended) An The air conditioner as recited claimed in claim 20, utilizing a nonflammable coolant whose having global warming potential is-lower than that of chlorofluorocarbon, wherein the coolant cooling/heating means includes:
- a second compressor for compressing a second coolant whose having an energy consumption efficiency is-higher than that of the <u>first</u> coolant;
- a second four-way valve for switching the-direction in which the second coolant as outputted output from the second compressor flows;
- a first heat exchanger for exchanging heat between the second coolant and outdoor air;
- a second flow control valve for regulating the flow volume of the second coolant; and a second heat exchanger for exchanging heat between the <u>first</u> coolant and the second coolant; wherein

when the air conditioner is being operated for cooling, the second coolant is circulated through the second compressor, the first heat exchanger, the second flow control valve, and the second heat exchanger, in that sequence, and

when the air conditioner is being operated for warming, the second coolant is circulated through the second compressor, the second heat exchanger, the second flow control valve, and the first heat exchanger, in that sequence.

- 22. (Currently Amended) An The air conditioner as recited claimed in claim 20, wherein the first compressor having has an intermediary-pressure inlet for drawing in the first coolant during compressing, the air conditioner further comprising:
- a thirdsecond flow control valve for regulating the flow volume of the first coolant inputtinginput into and outputtingoutput from the indoor heat exchanger;
 - a gas-liquid separator for separating into gas and liquid the first coolant; and
- a bypass pipe for introducing into the intermediary-pressure inlet at least part or all-of the gas of the coolant gas-separated by the gas-liquid separator; wherein

when the air conditioner is being operated for cooling, the <u>first</u> coolant is circulated through the <u>first</u> flow control valve, the gas-liquid separator, the <u>thirdsecond</u> flow control valve, and the indoor heat exchanger, in that sequence, and

when the air conditioner is being operated for warming, the <u>first</u> coolant is circulated through the indoor heat exchanger, the <u>thirdsecond</u> flow control valve, the gasliquid separator, and the <u>first</u> flow control valve, in that sequence.

- 23. (Currently Amended) AnThe air conditioner as recited claimed in claim 20, further comprising:
- a <u>thirdsecond</u> compressor for compressing the <u>first</u> coolant as compressed by the <u>first</u> compressor;
- a <u>thirdsecond</u> flow control valve for regulating <u>the</u> flow volume of the <u>first</u> coolant <u>inputtingingut</u> into and <u>outputtingoutput</u> from the indoor heat exchanger;
 - a gas-liquid separator for separating into gas and liquid the first coolant; and
- a bypass pipe for introducing into the thirdsecond compressor at least part or all-of the evolunt gas of the first coolant separated by the gas-liquid separator; wherein,

the <u>first</u> coolant <u>as outputted output</u> from the <u>third second</u> compressor is <u>inputted input</u> into the <u>first</u> four-way valve, <u>and</u>

when the air conditioner is being operated for cooling, the <u>first</u> coolant is <u>flowedflows</u> through the <u>first</u> flow control valve, the gas-liquid separator, the <u>thirdsecond</u> flow control valve, and the indoor heat exchanger, in that sequence, <u>meanwhile, and</u>

when the air conditioner is being operated for warming, the <u>first</u> coolant is <u>flowedflows</u> through the indoor heat exchanger, the <u>thirdsecond</u> flow control valve, the gasliquid separator, and the <u>first</u> flow control valve, in that sequence.

24. (Currently Amended) An The air conditioner as recited claimed in claim 20, further comprising:

- a third-radiator for radiating heat from the first coolant as outputted output from the first compressor; and
- a thirdsecond compressor for compressing the first coolant in a state in which after heat of the first coolant has been radiated away by the third radiator; and
- enditioner is being operated for warming.
- 25. (Currently Amended) An The air conditioner as recited claimed in claim 21, further comprising:
- a third compressor for compressing the <u>first</u> coolant as-compressed by the <u>first</u> compressor;
- a third heat exchanger for exchanging heat between the <u>first</u> coolant and the second coolant; and
- æ-flow-route changing means for flowing the <u>first</u> coolant as <u>outputtedoutput</u> from the compressor through the third heat exchanger and the third compressor, in that sequence, when the air conditioner is being operated for cooling, and into the third compressor when the air conditioner is being operated for warming; wherein the <u>first</u> coolant as <u>outputtedoutput</u> from the third compressor is <u>inputtedinput</u> into the <u>first</u> four-way valve, and the second coolant as <u>outputtedoutput</u> from the second heat exchanger <u>is-flowedflows</u> through the third heat exchanger and the second compressor, in that sequence.
- 26. (Currently Amended) AnThe air conditioner as recited claimed in claim 21, further comprising:
- a third compressor for compressing the <u>first</u> coolant as-compressed by the <u>first</u> compressor;
- a third heat exchanger for exchanging heat between the <u>first</u> coolant and the second coolant; and
- a forththird flow control valve for regulating the flow volume of the second coolant flowing in the third heat exchanger, wherein
- the coolant as <u>outputtedoutput</u> from the <u>first</u> compressor is <u>flowedflows</u> through the third heat exchanger, the third compressor, and the <u>first</u> four-way valve, in that sequence, and

part of the second coolant as outputted output from the first heat exchanger is flowed flows through the forth third flow control valve, the third heat exchanger, and the second compressor, in that sequence.

- 27. (New) The refrigerator as claimed in claim 14, further comprising at least one of pressure-measuring means for measuring pressure of the first coolant between exiting the first flow control valve and entering the first evaporator, and second temperature-measuring means for measuring temperature of the first coolant exiting the first flow control valve, wherein the flow-control-valve-entrance-temperature control-range determination means determines a control range of the temperature of the first coolant entering the first flow control valve, using the pressure of the first coolant measured by the pressure-measuring means or the temperature measured by the second temperature-measuring means.
- 28. (New) The refrigerator as claimed in claim 14, further comprising pressure-measuring means for measuring pressure of the first coolant between exiting of the first radiator and entering of the first flow control valve, wherein the flow-control-valve-entrance-temperature control-range determination means determines a control range of the temperature of the first coolant entering the first flow control valve, using the coolant pressure of the first coolant measured by the pressure-measuring means.